CDS 101/110 Homework #2 Solution

Problem 1 (CDS 101, CDS 110): (20 points for CDS 101, 25 Points for CDS 110)

(a) At equilibrium, $\ddot{q} = \dot{q} = 0$. So, $m\ddot{q} = -k(q + aq^3) - b\dot{q}$ becomes $0 = -k(q + aq^3)$. Solve for q to get q = 0. Ignore the imaginary q because this is a physical system.

Therefore, the equilibirum point is $(q, \dot{q}) = (0, 0)$. Note that q = 0 alone is not enough.

(b) Code:

```
m = 1000;
k = 250;
a = 0.02;
b = 100;
[x1,x2]=meshgrid(-1:0.2:1, -1:0.2:1);
xldot = x2;
x2dot = (-(k/m)*(x1+a*x1^3))-b*x2/m;
quiver(x1,x2,xldot,x2dot);
xlabel('$q$','Interpreter','latex')
ylabel('$\dot{q}$','Interpreter','latex')
```

Plot:



(c) Let $x = (x_1, x_2)$ where $x_1 = q$ and $x_2 = \dot{q}$. Rewrite $m\ddot{q} = -k(q + aq^3) - b\dot{q}$ as

$$\frac{dx}{dt} \triangleq F = \begin{bmatrix} x_2 \\ -\frac{k}{m}(x_1 + ax_1^3) - \frac{b}{m}x_2 \end{bmatrix}$$

Take the derivative:

$$\frac{dF}{dx} = \begin{bmatrix} 0 & 1\\ -\frac{k}{m}(1+3ax_1^2) & -\frac{b}{m} \end{bmatrix}$$

At equilibrium,

$$\frac{dF}{dx} \triangleq A \begin{bmatrix} 0 & 1\\ -\frac{k}{m} & -\frac{b}{m} \end{bmatrix}$$

Thus, the linearized system is

$$\frac{dx}{dt} = \begin{bmatrix} 0 & 1\\ -\frac{k}{m} & -\frac{b}{m} \end{bmatrix} x$$

- (d) The eigenvalues of A are $-0.05 \pm 0.4975i$. The real parts are all less than zero. Therefore, the system is assymptotically stable.
- (e) Solution to linear system can be rewrite as an exponential function. Since the real parts of all eigenvalues of A are less than zero, the system is decaying exponentially. Therefore, the system is exponentially stable.

Problem 2 (CDS 101, CDS 110): (15 points)

Code:

```
% x1 represents the reference velocity minus velocity
% x2 represents the velocity
[x1,x2]=meshgrid(-5:0.5:5, 15:0.5:25);
% Set the reference speed and the current error
Vr = 20;
Verr = Vr - x2;
% Gain values
Kp = .5; Ki = .1;
% Set the throttle
u = Kp*Verr + Ki*x1;
% Saturate the input of the throttle
u = min(u, 1); u = max(0, u);
% Parameters for defining the system dynamics (agree with simulink model)
m = 1000;
                       % mass of the vehcile
alpha = 16; % gear ratios
Tm = 190; % engine torque constant, Nm
wm = 420; % peak torque rate, rad/sec
```

```
beta = 0.4; % torque coefficient
Cr = 0.01; % coefficient of rolling friction
rho = 1.3; % density of air, kg/m?3
Cd = 0.32; % drag coefficient
A = 2.4; % car area, m?2
g = 9.8; % gravitational constant
qear = 3;
                        % choose the desired gear
theta = 0;
                        % the angle of the surface you?re driving on
% Determine the driving force
omega = alpha*x2;
torque = u .* Tm .* (1 - beta * (omega/wm - 1).^2);
F = alpha * torque;
% Determine the opposing forces
Fr = m \star g \star Cr;
                                    % Rolling friction
Fa = 0.5 * rho * Cd * A * (x2).^2; % Aerodynamic drag
Fg = m * g * sin(theta);
                                    % Road slope force
Fd = Fr + Fa + Fg;
                                    % total deceleration
% Determine the right side of the differential equation
dx1 = Verr;
dx2 = (F - Fd)/m;
% Plot
```

```
quiver(x1,x2,dx1,dx2)
ylabel('Velocity')
xlabel('Integral of reference velocity minus velocity')
```

Plot:



Problem 3 (CDS 110): (15 points)

(a) Clearly $V_1(x)$ is positive definite. To examine $\dot{V}_1(x)$, taking the derivative of V_1 and substituting the dynamics, we have

$$\frac{dV_1(x)}{dt} = -ax_1^2 - bx_1x_2 - cx_2$$

To check whether this is negative definite, we complete the square by writing

$$\frac{dV_1(x)}{dt} = -a(x_1 + \frac{b}{2a}x_2)^2 - (c - \frac{b^2}{4a})x_2^2$$

Since a > 0, clearly V_1 is negative semidefinite if $c - \frac{b^2}{4a} \ge 0$.

(b) Consider $V_2(x) = \frac{1}{2}x_1^2 + \frac{1}{2}(x_2 + \frac{b}{c-a}x_1)^2$. It is easy to show that $V_2(x)$ is positive definite since $V_2(x) \ge 0$ for all x and $V_2(x) = 0$ implies that $x_1 = 0$ and $x_2 + \frac{b}{c-a}x_1 = x_2 = 0$. Note that here we require that $c \ne a$.

We now check the time derivative of V_2 :

$$\frac{dV_2(x)}{dt} = x_1 \dot{x}_1 + (x_2 - \frac{b}{c-a} x_1)(\dot{x}_2 + \frac{b}{c-a} \dot{x}_1)$$
$$= -ax_1^2 + (x_2 + \frac{b}{c-a} x_1)(-bx_1 - cx_2 - \frac{ab}{c-a} x_1)$$
$$= -ax_1^2 + (x_2 + \frac{b}{c-a} x_1)(-cx_2 - \frac{bc}{c-a} x_1)$$
$$= -ax_1^2 + -c(x_2 + \frac{b}{c-a} x_1)^2$$

We see that $V_2(x)$ is negative definite. Hence, we show that $V_2(x)$ is a Lyapunov function as long as $c \neq a$.

Problem 4 (CDS 110): (15 points)

Elimination of u gives the following differential equation for the closed loop system

$$\frac{dx}{dt} = \begin{bmatrix} k & 1\\ -4k & -3 \end{bmatrix} x$$

This equation has the characteristic polynomial

$$\lambda(s) = (s-k)(s+3) + 4k = s^2 + (3-k)s + k$$

Solving for s by setting $\lambda(s) = 0$, we get $s = 0.5(k - 3 \pm \sqrt{9 + k^2 - 10k})$.

Code:

```
k = 0:0.01:10;
root_1 = 0.5*(k-3 + sqrt(9+k.^2-10*k));
root_2 = 0.5*(k-3 - sqrt(9+k.^2-10*k));
plot(real(root_1),imag(root_1),real(root_2),imag(root_2))
xlabel('Real')
ylabel('Imaginary')
```

Plot:

